

WINTER - 2022 EXAMINATION

Model Answer

Subject Name: Refrigeration & Air conditioning

Subject Code:

22660

Important Instructions to examiners:

- 1) The answers should be examined by key words and not as word-to-word as given in the model answer scheme.
- 2) The model answer and the answer written by candidate may vary but the examiner may try to assess the understanding level of the candidate.
- 3) The language errors such as grammatical, spelling errors should not be given more Importance (Not applicable for subject English and Communication Skills.
- 4) While assessing figures, examiner may give credit for principal components indicated in the figure. The figures drawn by candidate and model answer may vary. The examiner may give credit for any equivalent figure drawn.
- 5) Credits may be given step wise for numerical problems. In some cases, the assumed constant values may vary and there may be some difference in the candidate's answers and model answer.
- 6) In case of some questions credit may be given by judgement on part of examiner of relevant answer based on candidate's understanding.
- 7) For programming language papers, credit may be given to any other program based on equivalent concept.
- 8) As per the policy decision of Maharashtra State Government, teaching in English/Marathi and Bilingual (English + Marathi) medium is introduced at first year of AICTE diploma Programme from academic year 2021-2022. Hence if the students in first year (first and second semesters) write answers in Marathi or bilingual language (English +Marathi), the Examiner shall consider the same and assess the answer based on matching of concepts with model answer.

| 1 | | Attempt any <u>FIVE</u> of the following: | 10 Marks |
|---|------|---|----------------------------|
| | (a) | Give unit of Refrigeration and define it. | |
| | Ans. | Unit of refrigeration is Ton. | |
| | | One Ton of refrigeration: A ton of refrigeration is defined as the quantity of heat required to be removed to from one ton of ice at 0^0 C in 24 hours when initial condition of water is 0^0 C 1 Ton of refrigeration = 3.517 KJ/Sec or 3.517 kW | 02 |
| | (b) | Give any two excellent properties of NH3, used as refrigerants. | |
| | Ans. | i) High Latent heat of vaporization ii) Low mass flow rate per ton of refrigeration iii) Leaks can be easily detected because of its strong smell. iv) Easily liquefied on applying pressure even at room temperature v) It is safe to environment & does not cause depletion of the ozone layer. vi) It is cheap and easily available | Any Two Points 02 |



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| Subject Nam | e: Refrigeration & Air conditioning <u>Subject Code:</u> 22660 | D |
|-------------|--|----------------------|
| (c) | List parameters controlled in 'air-conditioning' system. | |
| Ans. | i) Temperature of air ii) Humidity of air iii) Motion of air iv) Purity of air | 1⁄2 Each point |
| (d) | A refrigerator works on reversed Carnot cycle between the temperature limits of -5 ^o C and 35 ^o C. Find out its COP. | |
| Ans. | Given:- $T_L = -5^0 + 273 = 268 \ ^0K$ $T_H = 35^0 + 273 = 308 \ ^0K$ COP = ? We know that $(COP)_{Carnot} = T_L / T_H - T_L$ | 01 |
| | = 268 / 308-268 | |
| | = 6.8 | 01 |
| (e) | Give classification of ducts | |
| Ans. | The ducts may be classified as follows: i) According to type of air in duct: Supply duct Return duct Fresh air duct ii) According to pressure of air in duct: Low pressure duct Medium pressure duct High pressure duct iii) According to velocity of air in duct: Low velocity duct High velocity duct | Any two |



Enlist advantages of Hermetically sealed compressor over open type compressor (f) there is no chance of leakage of costly refrigerant i) Ans. Any Four ii) high efficiency and reliability of the compressor motor Points 02 iii) Less noise & vibration iv) Require small space because of compactness v) Do not need any shaft seal assembly, because the compressor and the motor are mounted on a common shaft and in a common housing. vi) Less maintenance and long life On p-H and T-S diagram shown vapour compression cycle started compression with (g) wet inlet and dry outlet. Ans. ρ Т condh ス condr 3 02 Expansion co. Comp Exph Evaph 4 Evapn 1 H ►G VCC on p-H & T-S diagram.







| | | i) Superheating causes increase in specific volume at suction of compressor which reduces capacity of compressor ii) Superheating increase refrigerating effect, at the same time it increases the amount of workload supplied to compressor. Hence COP of cycle remain unaffected or in some cases it reduces | Explanation 02 |
|---|--------------|---|-------------------|
| | (c) | Give important desirable properties of an 'Ideal refrigerant' | |
| | Ans. | 1) Boiling point at atmospheric pressure should be low. | Any 8 points |
| | | 2) Freezing point at atmospheric pressure should be low. | |
| | | 3) Latent heat of vaporization of refrigerant must be high. | 04 |
| | | 4) Critical temperature should be high. | |
| | | 5) It should not have corrosive action with system material. | |
| | | 6) It should not be flammable & explosive. | |
| | | 7) It should not be toxic. | |
| | | 8) It leak should be easily detectable. | |
| | | 9) It should have positive condensing pressure. | |
| | | 10) It should have satisfactory heat transfer coefficient. | |
| | | 11) It should have high thermal conductivity. | |
| | | 12) It should have chemical stability. | |
| | (d) | For 'storage tank type water cooler' suggest - | |
| | Ans. | i) Compressor - Hermetically sealed compressor | |
| | | ii) Condenser – Force convection air cooled condenser | |
| | | iii) Expansion device – Capillary tube & thermodynamic expansion valve | 04 |
| | | iv) Evaporator – Dry expansion type evaporator | |
| | | v) Refrigerant – R-134-a or R-22 | |
| 3 | | Attempt any <u>THREE</u> of the following: | 12 |
| | | | IVIarks |
| 3 | a) | Enlist factors affecting human comfort. | |
| | Ans | 1. Effective temperature | |
| | | 2. Heat production and regulation in human body | |
| | | 3. Heat and moisture losses from the human body | |
| | | 4. Moisture content of air | ½ Marks |
| | | 5. Quality and quantity of air | each |
| | | 6 Air motion | |
| | | | |



| | | 7. Hot and cold surfaces | |
|---|-----------|---|------------------------------|
| | | 8. Air stratification | |
| | b) | Explain working of 'Flooded Type Evaporator' with a neat sketch. | |
| | Ans: | Flooded type of evaporator feeds excess of liquid refrigerant so that the exit of the evaporator will be a mixture of liquid and vapor refrigerant. | Labeled Sketch 2 Marks |
| | | Working: | |
| | | In the flooded type of evaporator coil remains completely filled with liquid refrigerant as | |
| | | shown in the figure. The level of liquid refrigerant is maintained constant in the surge | |
| | | chamber by using float control. The liquid refrigerant enters into evaporator coil from the | |
| | | surge chamber. In evaporator coil, part of liquid refrigerant boils and converts into vapor. | Marking. |
| | | The vapor formed is collected at the top of the surge chamber and the remaining liquid | working |
| | | refrigerant is returned to the surge chamber. From the top of the surge chamber, refrigerant | 2 Marks |
| | | vapors are drawn in the suction line of the compressor. In the flooded type evaporator rate | |
| | | refrigerant but this type of refrigerant requires a large amount of refrigerant | |
| 3 | c) | Enlist pressure losses occurred in the duct | |
| 5 | C) Ame | 1) Frictional Losses | |
| | AU2 | The pressure is lost due to friction between the moving particles of the fluid and the | |
| | | interior surfaces of a duct. This is termed as friction loss. | 1 Marks |
| | | 2) Dynamic Losses: | |



| | | Pressure is also lost dynamically at the changes of direction such as in bends, | 3 Marks |
|---|-----|--|------------------------------|
| | | elbows, etc. and at the cross-section changes of the duct. This is termed as dynamic | ½ mark for |
| | | loss. | each |
| | | i) Pressure Loss in Elbows | Dynamic |
| | | ii) Loss due to enlargement | loss |
| | | iii)Loss due to contraction | |
| | | iv) Losses at suction and discharge openings | |
| | | v) Pressure losses in fittings (valves, grills and others) | |
| 3 | d) | Explain working of Thermostatic Expansion Device with neat sketch. | |
| | Ans | Capillary tube Capillary tube | Labeled Sketch 2 Marks |
| | | Working: | |
| | | • The thermostatic expansion valve consists of a needle valve, a seat, a metallic | |
| | | diaphragm, a spring, adjusting screw and a feeler bulb. | |
| | | • The opening or closing of valve depends upon following forces acting on the | |
| | | diaphragm: | |
| | | i. Spring pressure acting on bottom of diaphragm Ps | |
| | | ii. Evaporator pressure acting on bottom of diaphragm P_E | |
| | | iii. Feeler bulb pressure acting on top of diaphragm P_B | |
| | | • If load on evaporator increases, it causes the liquid refrigerant to boil faster in | |
| | | evaporator coil. Since feeler bulb is installed on the suction line, therefore it is at | |
| | | the same temperature as refrigerant at that point. So temperature of the bulb | |
| | | increases due to early vaporization of refrigerant. | |
| | | • Thus the feeler bulb pressure increases and gets transmitted through the capillary | |



| | | tube to the diaphragm. The diaphragm moves downwards, opening the valve to admit more liquid refrigerant into the evaporator | |
|---|------|--|--------------------|
| | | • This continues till pressure equilibrium on diaphragm is reached at which feeler | |
| | | • This continues the pressure equinoritant on diaphragin is reached, at which receipt | |
| | | pressure acting at top of diaphragm is balanced by spring and evaporator | Working |
| | | • When evaporator load decreases less liquid refrigerant evaporates in the coil and | 2 Marks |
| | | the excess liquid flows towards the outlet. This cools the feeler hulb and its pressure | |
| | | and temperature decreases | |
| | | This pressure makes the disphram move unward reducing the value opening and | |
| | | • This pressure makes the draphragin move upward, reducing the varve opening and | |
| | | in turn decreasing reirigerant flow to evaporator. This causes decrease in evaporator | |
| | | pressure and again continues till diaphragm pressure equilibrium is reached. | |
| | Ans: | A close of matrice of particle of a rand 4.2 bar. The air is cooled in cooler to a temperature of 50 °C and temperature of air at inlet to compressor is -20 °C. Determine i) COP of the cycle ii) Mass of air circulated per minute. Given: Q=10 TR; P_1=P_2=1.4 bar; P_3 = P_4 = 4.2 bar; T_2 = -20°C = 253K T_4=50°C=323K For ideal gas $C_P = 1.07$ kJ/kg-K (for dense air) $4.2 \int_{DOT} \frac{4.2}{1} \int_{DOT} \frac{1}{1} \int_{OT} \frac{1}{2} \int_{OT} \frac{1}{2} \int_{OT} \frac{1}{2} \int_{OT} \frac{4.2 \text{ bor}}{1} \int_{OT} \frac{4.2 \text{ bor}}{1} \int_{OT} \frac{4.2 \text{ bor}}{1} \int_{OT} \frac{1}{1} \int_{OT} \frac$ | 2 Marks 2 Marks |
| 4 | | Attempt any THREE of the following: | 12 |
| | 1 | Page No: | / N |



| a) An | Explain Frosting of evaporator. On this basis classify evaporators. | |
|----------|---|------------|
| | Frosting of evaporator: | Frosting |
| | The condensation of water vapour of the room/cold storage causes formation of frost over | 2 Marks |
| | the evaporator. | |
| | Formation of ice takes place in all the evaporators which are operating below the freeing point of water (0 °C). | |
| | The accumulation of ice over the heat transfer surface reduces the heat transfer rate as the | |
| | ice is poor conductor of heat. Therefore, it is necessary to remove the ice deposited over the evaporator at periodic time interval. | |
| | The operation of removing frosted ice from the evaporator is known as defrosting of | |
| | evaporator. The period of defrosting depends on type of evaporator, relative humidity of | Classifica |
| | the cold room, evaporation temperature etc. | 2 marks |
| | Classification of Evaporators based on Frost i) Frosting Evaporators ii) Non-Frosting Evaporators iii) Defrosting Evaporators | |
| An | heating coil. | 2 Marks |
| | DB1 DB2 Dry Bulb Temperature (degree C) OB2 By-Pass Factor: It is defined as the ratio of loss in cooling or heating to the ideal cooling or heating. | |



| | | It is denoted by X. | |
|---|----------|---|-----------|
| | | Bypass factor (X) for heating coil = Loss in heating / Ideal heating | |
| | | The bypass factor (B.P.F.) in case of sensible heating of air is | |
| | | = td3-td2 / td3-td1 Where, | 1 Mark |
| | | $td_1 = Dry$ bulb temperature of air entering the heating coil, | |
| | | $td_2 = Dry$ bulb temperature of air leaving the heating coil and $td_2 = Dry$ bulb temperature of heating coil | |
| | | tu ₃ = Dry buib temperature of neating con | 1 Mark |
| 4 | c) | Enlist insulating materials used in refrigeration field. | Any four |
| | Ans | i) Cork | 1 mark |
| | | ii) Mineral Wool | each |
| | | iii) Fiber/Cellular glass or glass wool, | |
| | | iv) Cellulose | |
| | | v) Polyuretnane Foam (PU Foam) vi) Polystyrene or thermocole | |
| | | vij) Forystytene of thermocole vii) Gypsum | |
| | | viii) Aluminium Foils | |
| | | ix) Vermiculite | |
| _ | | x) Blast Furnace slag | |
| 4 | d) | Explain working of 'Vapor Compression Cycle' with block diagram. Plot it on p-h and T-S diagram | |
| | | | |
| | A | | |
| | Ans: | | |
| | | ¥ + 2 | Block dia |
| | | Condenser | 1 Mark |
| | | | TIMALK |
| | | 3 High Pressure | |
| | | Side | |
| | | Expansion | |
| | | Device Compressor | |
| | | | |
| | | | |
| | | | |
| | | Low Pressure | |
| | | - 4 Side | |
| | | | |
| | | Evaporator | |
| | | | |
| | | Figure: Vapor Compression Cycle | |
| | | | |
| | | | |
| | | | |
| | | | |
| | | | |



Q. 5

А

25

298.9

1166.94

1.1242

05







Page No: ___/ N







| | | Q=U*A*te | |
|---|-----|---|-----------|
| | | Where, | |
| | | Q=Total heat transfer, | |
| | | A=area of roof or wall, | |
| | | te = Equivalent temperature differential. | |
| | | 3. Heat gain due to infiltration (using air change method) | |
| | | Amount of infiltrated air through windows and wall is | |
| | | = $(L^*W^*H^*Ac)/60 \text{ m}^3/\text{min}$. Both sensible and latent heat load gain. | |
| | | 4. Heat gain through ventilation | |
| | | The ventilation (supply of outside air) is provided to the conditioned space in order to | |
| | | buildings in normal ceiling heights. The outside air adds sensible as well as latent heat | |
| | | 10au. 5. Haat gain from annliances/lightening aguinment's | |
| | | 5. Heat gain from apphances/fightening equipment s – | |
| | | Q = (Total Wattage *use factor*Allowance Factor). | |
| | | 6. Heat gain from Occupants | |
| | | The amount of heat dissipated would depend on the number of persons and their activities, | |
| | | age, sex, cloths. | |
| | | Heat gain depends on average number of people present in Auditorium. | |
| | | Q= (no of persons)*(load per person). | |
| 6 | | Attempt any TWO of the following: | 12 |
| | а | Explain 'Air Refrigeration System' used for aircraft with block diagram. Also | 2 M |
| | | represent it on T-S diagram and find out its COP. | (Sketch) |
| | | | , |
| | Ans | Gas turbine Main comp. Bam + Ambient air | |
| | | | |
| | | Air to | |
| | | | |
| | | Combustion Cooling | |
| | | chamber | |
| | | | |
| | | | |
| | | | |
| | | Heat | |
| | | exchanger Cooling | |
| | | | |
| | | To atmosphere | |
| | | Simple air cooling system. | |
| | | The main components of air cooling system are the main compressor driven by a gas | |
| | | turbine, a heat exchanger, a cooling turbine and a cooling air fan. The air required for | |
| | | refrigeration system is bled off from the main compressor. This high pressure and high | |
| | | temperature air is cooled initially in the heat exchanger where ram air is used for cooling. It | |
| | | is further cooled in the cooling turbine by the process of expansion. The work of this | 2 M |
| | | turbine is used to drive the cooling fan which draws cooling air through the heat exchanger. | (Working) |
| | | The various process consist of this system are: | 0, |
| | | · · · | |







(ISO/IEC - 27001 - 2013 Certified)



The simple absorption system is not very economical. In order to make the system more practical, it is fitted with an analyzer, a rectifier and two heat exchangers as shown in fig. These accessories help to performance and working of the plant, as discussed below:

1.Analyser

When ammonia is vaporized in the generator, some water is also vaporized and will flow into the condenser, along with the ammonia vapours in the simple system. If these unwanted water particles are not removed before entering into the condenser, they will enter into the expansion value where they freeze and choke the pipeline. In order to remove these unwanted particles flowing to the condenser, an analyzer is used. The analyzer may be built as an integral part of the generator or made as a separate piece of equipment. It consists of a series of trays mounted above the generator. The strong solution from the absorber and the aqua from the rectifier are introduced at the top of the analyzer and flow downward over the trays and into the generator. In this way, considerable liquid surface area is exposed to the vapour rising from the generator. The vapour is cooled and most of the water vapour condenses, so that mainly ammonia vapour (approximately 99%) leaves the top of the analyzer. Since the aqua is heated by the vapour, less external heat is required in the generator.

2. Rectifier

In case the water vapours are not completely removed in the analyzer, a closed type vapour cooler called rectifier (also known as dehydrator) is used. It is generally water cooled Its function is to cool further the ammonia vapours leaving the analyzer so that remaining water vapours are condensed. Thus, only dry or anhydrous ammonia vapours flow to the condenser. The condensate from the rectifier is returned to the top of the analyzer by a drip return pipe.

3. Heat exchangers

2 M-Description



6





through a filter to remove dirt, dust and other impurities. The air now passes through cooling coil. The coil has a temperature less than dew point temperature of the air. When the air is cooled and passed through a membrane, some quantity of water vapours associated with air is condensed due to cooling of air below its dew point temperature. Thus the air loses the moisture in the form of condensate and is collected in sump. Due to cooling below DPT, the temperature of air decreases below the desired temperature. Therefore this air is heated with the help of heating coil. This is done to bring the air to desire DBT and RH. The air washer system is not operative in this case of summer air conditioning system. Now conditioned air passes to conditioned space by a fan. From the conditioned space the part of air is exhausted to atmosphere by exhaust fans or ventilators. The remaining part of the used air or recirculated air is again conditioned as shown in the figure. The outside air is sucked and made to mix with recirculated air in order to make up for the loss of conditioned or used air through exhaust fans or ventilators from conditioned space. 3M-for Sketch Recirculated air Conditioned space Perforated Outside Air membrane air washer (Fresh atmospheric air) Air damper Fan Heating coil Filter Cooling coil Cold water Sump (water reservoir) Make up Water circulating pump water -----END-----