

MAHARASHTRA STATE BOARD OF TECHNICAL EDUCATION (Autonomous)

(ISO/IEC - 27001 - 2013 Certified)

SUMMER - 2022 EXAMINATION

Subject Name: Theory of Structure

Model Answer

Subject Code:

22402

Important Instructions to examiners:

- 1) The answers should be examined by key words and not as word-to-word as given in the model answer scheme.
- 2) The model answer and the answer written by candidate may vary but the examiner may try to assess the understanding level of the candidate.
- 3) The language errors such as grammatical, spelling errors should not be given more Importance (Not applicable for subject English and Communication Skills.
- 4) While assessing figures, examiner may give credit for principal components indicated in the figure. The figures drawn by candidate and model answer may vary. The examiner may give credit for any equivalent figure drawn.
- 5) Credits may be given step wise for numerical problems. In some cases, the assumed constant values may vary and there may be some difference in the candidate's answers and model answer.
- 6) In case of some questions credit may be given by judgement on part of examiner of relevant answer based on candidate's understanding.
- 7) For programming language papers, credit may be given to any other program based on equivalent concept.
- 8) As per the policy decision of Maharashtra State Government, teaching in English/Marathi and Bilingual (English + Marathi) medium is introduced at first year of AICTE diploma Programme from academic year 2021-2022. Hence if the students in first year (first and second semesters) write answers in Marathi or bilingual language (English +Marathi), the Examiner shall consider the same and assess the answer based on matching of concepts with model answer.

Q. No.	Sub Q. N.	Answer	Marking Scheme
Q-1		Attempt any FIVE of the following:	10 M
	a)	Define core of the section.	2 M
	Ans.	Core of a section: Core of the section is that portion around the centroid in within which the line of action of load must act, so as to produce only compressive stress is called as core of the section. OR It is also defined as the region or area within which if load is applied, produces only	1 M
		compressive resultant stress. OR If Compressive load is applied, the there is no tension anywhere in the section. $e_{max} = d/8$ e = Core of section for Section For Circular section	01 M



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	b)	Give relationship between bending moment, slope and deflection.	
	Ans.	$\frac{d_2 y}{dx^2} = \frac{M}{EI}$	01 M
		Where, θ = Slope at any section Y= Deflection of Beam M= Bending Moment E=Modulus of Elasticity I=Moment of Inertia	01 M
	c) Ans.	State the effect of continuity in continuous beam.Solution:Effect of continuity: If a beam is continuous, over the support, a hogging moment developed at that support which tries to bring the beam back to its equilibrium condition	01 M
		 as it was before loading. Thus the beam deflection and consequently the load carryin of the is increased. Effects of continuity are as follows. i) Produces support moment of hogging nature. ii) Reduces bending moment along the span. iii) Reduces deflection and increases load carrying capacity. iv) Sagging moment occurs at mid span. 	g
		$ \begin{array}{c} $	01 M
		AT AT AT	
	d)	Define	
	Ans.	 i) Carry over factor ii) Stiffness factor i) Carry over factor: It is the ratio of moment produced at a joint to apply at the other end of the member. it is (1/2). 	er 01 M
		ii) Stiffness factor: It is (1/2).ii) Stiffness factor: It is the moment required to obtain unit rotation at an end without translating it.	ut 01 M



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Subject Name: Theory of Structure **Model Answer** Subject Code: 22402 With sketch, state the different types of portal frame. 02 M e) i. Symmetrical portal frame (Non sway type) 01 M Ans. ii. Unsymmetrical portal frame (Sway type) Sway mode Non-sway mode 01 M (Fig.) 02 M f) State the middle third rule. Middle third rule: In case of rectangular cross section, if the load is applied at location Ans. along the middle third part of both mutually perpendicular axes then the stresses 01M produced compressive nature. d d/3 01M b b For Rectangular section Identify nature of support if g) i) $\theta = 0$, y = 0 ii) $\theta = 0, y \neq 0$ i) $\theta = 0, y = 0$ -----Fixed Support 01 M Ans. ii) $\theta = 0, y \neq 0$ ------Center of the Beam(Student May Right Propped Support) 01 M

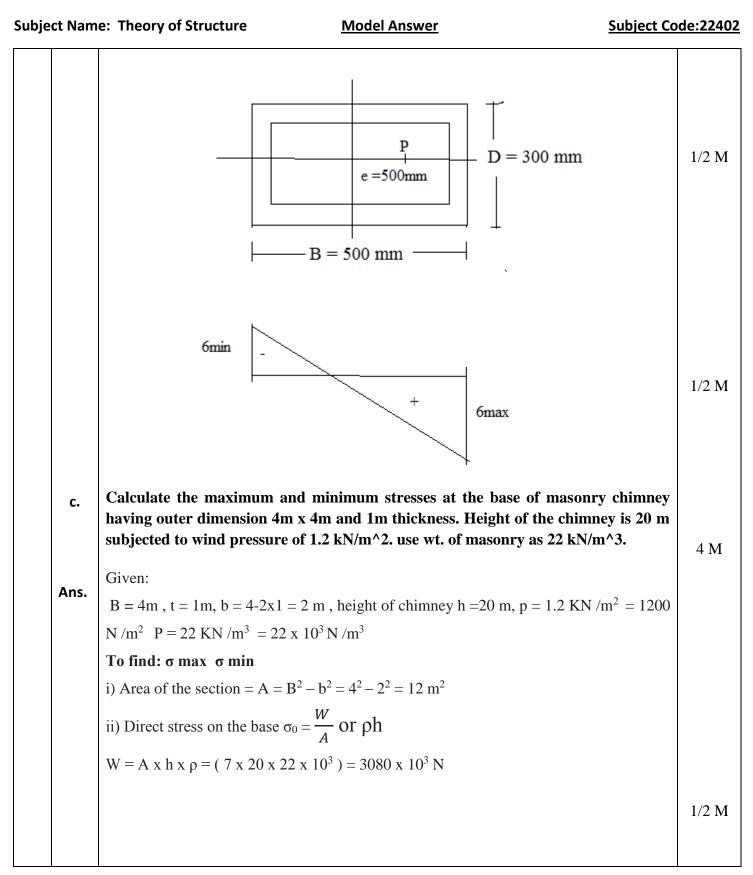


			46
2-2		Attempt any FIVE of the following:	12 M
	a)	Derive the expression for limit of eccentricity for rectangular section (b x d) $\frac{1}{2}$	4 M
		dimensions. Solution:	
		Let us consider a rectangular section of width b and thickness d as shown in fig.	
	Ans.		
		Area of section, $A = b x d$	
		$Zxx = \frac{I_{xx}}{Y_{max}} = \frac{\frac{bd^3}{12}}{\frac{d}{2}} = \frac{bd^2}{6}$	1/2 M
		$Z_{XX} = \frac{Y_{XX}}{Y_{max}} = \frac{T_{Z}}{\underline{d}} = \frac{T_{Z}}{6}$	
		2	
		$I_{\rm WV} = \frac{db^3}{12} db^2$	
		$Zyy = \frac{I_{yy}}{Y_{max}} = \frac{\frac{db^3}{12}}{\frac{b}{2}} = \frac{db^2}{6}$	1/2 M
		2	
		For no tension condition,	
		$e \leq \frac{Zxx}{A}$ and $e \leq \frac{Zyy}{A}$	
			1/2 N
		$e \le \frac{\frac{bd^2}{6}}{bd}$ and $e \le \frac{\frac{db^2}{6}}{bd}$	1/2 N
		$e \leq \frac{1}{bd}$ and $e \leq \frac{1}{bd}$	
		$e \le \frac{d}{6}$ and $e \le \frac{b}{6}$	1/2 M
		ie $e_x = \frac{d}{6}$ and $e_y = \frac{b}{6}$ $2 e_x = \frac{d}{3}$ and $2e_y = \frac{b}{3}$	1 M
		0 0 5 5	1 101
		× ×	
		- b	
			1/2 N
		$d x = \frac{2e_x}{2e_x}$	
		$d/2$ $1 \rightarrow e_{y} \rightarrow i = 3$	
		- b/3 -→ b/3 -→ b/3-→	
		h	
		1	



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b)	Calculate intensity of stresses at base of hollow rectangular column 500mm x 300mm external dimension and 50mm thickness subjected to vertical load of l60kN with 200mm eccentricity parallel to 500mm side of column.	4 M
Ans.	Given: P= 160 KN = 160 x 10^{3} N e = 500 mm Outer dimensions: B = 500 mm, D = 300 mm Inner dimensions: t = 50 mm b = 500 -100 = 400 mm d = 300 - 100 = 200 mm	
	Area, A = BD - bd = $(500 \text{ x } 300) - (400 \text{ x } 200) = 70000 \text{ mm}^2$ Section modulus, I $\frac{DB^3}{db^3}$	1/2 M
	$Zyy = \frac{I_{yy}}{Y_{max}} = \frac{\frac{DB^3}{12} - \frac{db^3}{12}}{\frac{B}{2}}$ $Zyy = \frac{DB^3 - db^3}{6B}$ $Zyy = \frac{300 \times 500^3 - 200 \times 400^3}{6 \times 500}$	1/2 M
	$Zyy = \frac{160 \times 10^{6}}{6 \times 500}$ $Zyy = 8.23 \times 10^{6} \text{ mm}^{3}$ Direct stress, $60 = \frac{P}{A} = \frac{160 \times 10^{3}}{7000}$	1/2 M 1/2 M
	$= 2.285 \text{ N/mm}^{2}$ Bending stress, 6b = $\frac{M}{Zyy} = \frac{\text{P.e}}{Zyy} = \frac{160 \times 10^{3} \times 200}{8.23 \times 10^{6}}$ $= 3.886 \text{ N/mm}^{2}$	1/2 M
	$= 3.880 \text{ N/IIIII}^{-1}$ $6_{\text{max}} = 60 + 6b = 2.285 + 3.888 = 6.173 \text{ N/mm}^2 \text{ (Compressive)}$ $6_{\text{min}} = 60 - 6b = 2.285 - 3.888 = -1.603 \text{ N/mm}^2 \text{ (Tensile)}$	1/2 M







Subject N	Name: Theory of Structure <u>Model Answer</u> <u>Subje</u>	<u>ect Code:</u> 22402
	$\sigma_0 = \frac{3080 \text{ x} 10^3}{7}$	1/2 M
	= 440 x 10^3 N/m ² = 440 KN/m ² Or 20 x 22x 10^3 = 440 x 10^3 N/m ² = 440 KN/m ²	
	iii) Total wind load $P = C \times P \times P$ projected area	1/2 M
	= C x Þ x D x h = 1 x 1200 x 4 x 20 = 96000 N = 96 kN	1/2 M
	iv) Moment on the base M= P x $\frac{h}{2}$ = 96000 x $\frac{20}{2}$ =960 x 10 ³ Nm = 960 kNm v) Section modulus about Y-Y axis:	1/2 M
	$Zyy = \frac{I_{yy}}{Y_{max}} = \frac{B^4 - b^4}{6 x b} = \frac{4^4 - 2^4}{6 x 4}$	
	$Zyy = 10 \text{ m}^3 = 10x \ 10^9 \text{ mm}^3$	
	vi) Bending stress on the base section,	
	$\sigma b = \pm \frac{M}{Z} = \frac{960 \ x \ 10^3}{10} \text{ or } \frac{960}{10}$	
	$= \pm 96 \text{ x } 10^3 \text{ N/m}^2 = \pm 96 \text{ kN/m}^2$	1/2 M
	$\sigma_{max} = \sigma_0 + \sigma b = 440 + 96 = 536 \text{ kN/m}^2 \text{ (Comp)}$	1/2 M
	$\sigma_{\min} = \sigma_0 - \sigma b = 440 - 96 = 344 \text{ kN/m}^2 \text{ (Comp)}$	
	$6max = 536 \text{ KN/m}^2$	1/2 M



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d.	A short column of external diameter 250mm and internal diameter 200mm carries an eccentric load. Find the eccentricity which the load can have without producing section (Compression) in the section of column. Solution:	4 M
Ans.	D = 250 mm, d = 200 mm	
	A = $\frac{\pi}{4}$ (D ² - d ²) = $\frac{\pi}{4}$ (250 ² - 200 ²) = 17671.458 mm ²	
	$I = \frac{\pi}{64} (250^4 - 200^4) = 113.207 \text{ x } 10^6 \text{ mm}^4$	1/2 M
	$Z = \frac{I}{Y} = \frac{I}{\frac{D}{2}} = \frac{113.207 \text{ x } 10^6}{\frac{250}{2}} = 905.656 \text{ x } 10^3 \text{ mm}^3$	1/2 M
	2 2 Direct stress $\sigma_0 = \frac{P}{A} = \frac{P}{17671.458}$	1/2 M
	Bending stress, $\sigma_b = \frac{M}{Z} = \frac{P \times e}{Z} = \frac{P \times e}{905.656 \times 10^3}$	01 M
	For no tension condition, we know that	
	$\frac{\sigma_{o} = \sigma_{b}}{\frac{P}{17671.458}} = \frac{P x e}{905.656 x 10^{3}}$	1/2 M
	e = 51.249 mm	
		01 M

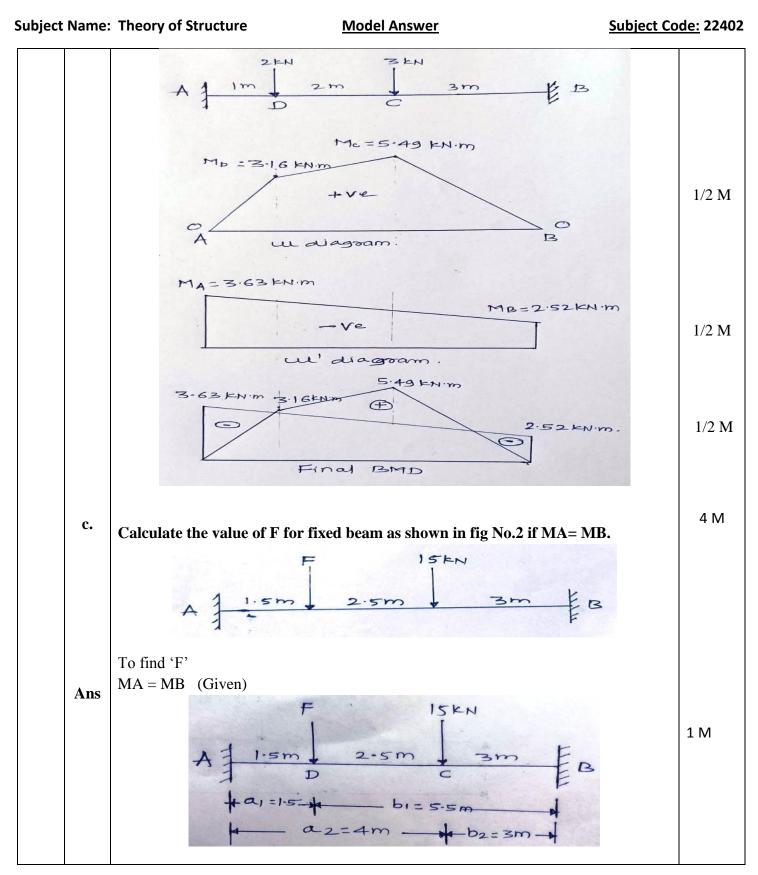


12	Attempt any THREE of the following	12 M
2-3	Using Macaulay's method calculate slope under point load of 15 KN acting at 3 m	4 M
a	from left hand support of simply supported beam of span 5 m in terms of EI.	4 101
	A X B	
	RAH X RB	
	$\Sigma F v = 0$, $R_A + R_B = 15$	
A	^{IS} M@A = 0, RB X 5 =15 X 3,	
	RB = 9 KN, RA = 6 KN	
	Consider section at x-x distance from Support A	
	$Mx = R_A \cdot x X - 15 (X-3)$	
	But Mx = EI $\frac{d^2 y}{dx^2}$	
	$d^2 v$	1/2 N
	EI $\frac{d^2 y}{dx^2}$ = 9 X – 15 (X-3) Equation 1	1/2 N
	Integrating Equation 1, we get	1/2 N
	EI $\frac{dy}{dx}$ = 9 X ² /2 - 15 (X-3) ² /2 + C ₁ slope equation 2	1/2 N
	Integrating Equation 2, we get	
	EI y = $9 X^{3}/6 - 15 (X-3)^{3}/6 + C_{1} X + C_{2}$ equation 3	
	To calculate C_2 apply boundary condition put x = 0, y = 0 in equation 3 , we get	1/2 N
	$0 = 0 + 0 + C_2$	
	$C_2 = 0$	01 N
	To calculate C_1 , apply boundary condition, at $x = 5$, $y = 0$ in equation 3	
	$0 = 6 \times \frac{5^3}{6} - 15 \frac{(5-3)^3}{6} + C_1 \times 5 + 0$	1/2 I
	$C_1 = -21$	
	To calculate slope under point load, put $x = 3m$ in slope equation 2, we get	
	$EI\frac{dy}{dx} = 6 \times \frac{3^2}{2} - 15 \times \frac{(3-3)^2}{2} - 21 = 6$	
	$\frac{dy}{dx}$ = 6/EI Slope under point load.	1/2 N



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b.	Calculate fixed end moments and draw BMD for fixed beam shown in fig No.1	4 M
	$\begin{array}{c} 2EN \\ \hline 3EN \\ \hline 1m \\ \hline 2m \\ \hline 3m \\ \hline B \\ \hline C \\ \hline B \\ \hline \end{array}$	
Ans	Step 1) To find support reaction and free bending moment by assuming beam to	
	be simply supported.	
	A IM 2m 3m B A IM 2m 3m B VA=100 NN Sil7 EN. Calculation of Readton	
	Support reaction $\Sigma F_y = V_A - 2 - 3 + V_B$	
	$V_A + V_B = 5$	
	$M @A = 0 = 2X1 + 3X3 - V_B X 6$	
	$V_{\rm B} = 1.83 \ {\rm KN}$	
	V _A =3.17 KN	
	Free bending Moments	
	$M_A = 0$, $M_B = 0$ Simply supported end	1/2 M
	$M_D = 3.17 \text{ X1} = 3.17 \text{ kN.m}$ (Sagging)	
	$M_{\rm C} = 1.83 \text{ X } 3 = 5.49 \text{ kN.m} \text{ (Sagging)}$	01 M
	Draw μ diagram using M _C and M _D Step 2) To find fixed end moments M _A & M _B	01 101
	$M_{\rm A} = \frac{-W_{1.}a_{1.}b_{1}^{2}}{L^{2}} - \frac{W_{2.}a_{2.}b_{2}^{2}}{L^{2}} = \frac{-2x1x5^{2}}{6^{2}} - \frac{3x3x3^{2}}{6^{2}} = -3.63 \text{ KN.m}$	
	$M_{\rm B} = \frac{-W_1.b_1.a_1^2}{L^2} - \frac{W_2.b_2.a_2^2}{L^2} = \frac{-2x5x1^2}{6^2} - \frac{3x3x3^2}{6^2} = -2.52 \text{ KN.m}$	1/2 M
	Draw μ ' diagram using M _A and M _B	
	By superimposing μ diagram and μ ' diagram, we get final BMD (As shown in fig)	1/2 M

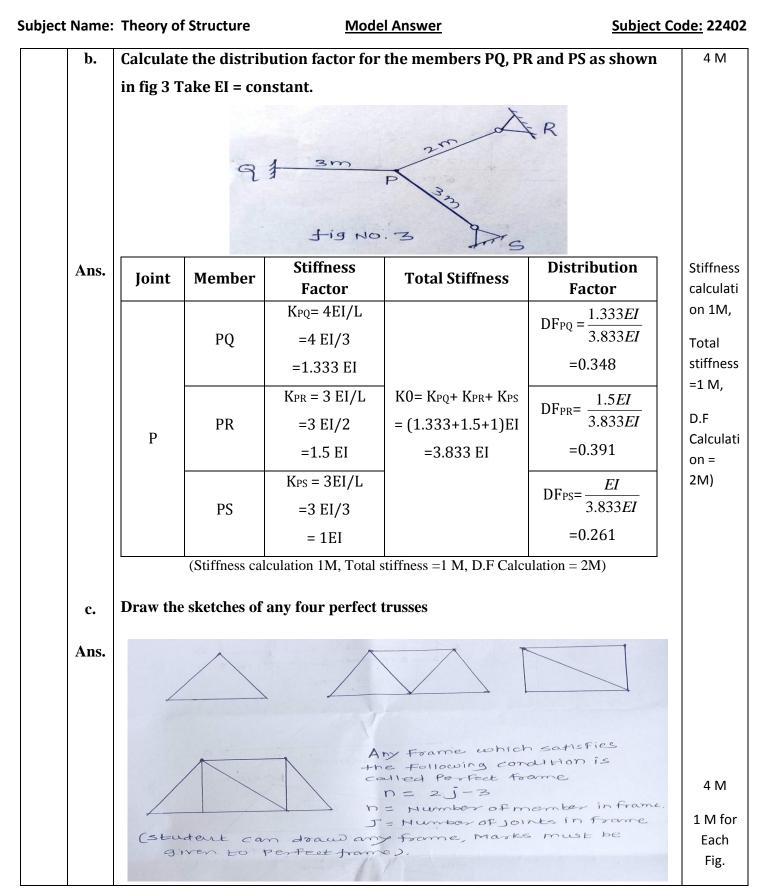




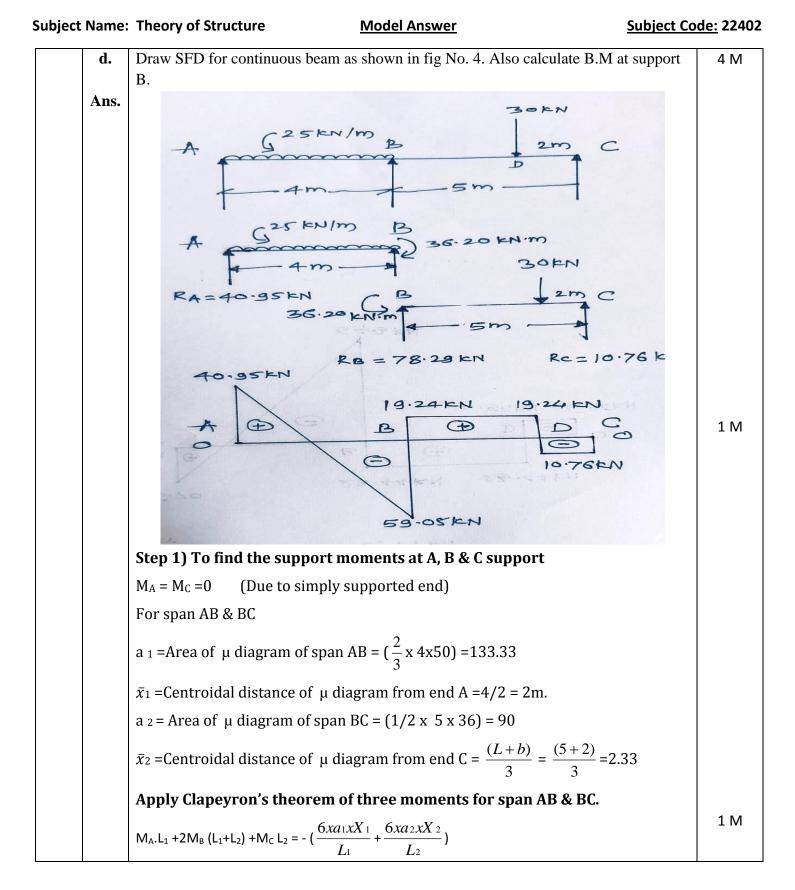


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		$M_{A} = \frac{-W_{1.}a_{1.}b_{1}^{2}}{L^{2}} - \frac{W_{2.}a_{2.}b_{2}^{2}}{L^{2}} = \frac{-Fx1.5x5.5^{2}}{7^{2}} - \frac{15x4x3^{2}}{7^{2}} = -0.92F-11.02 $ (1)	01 M
		$\mathbf{M}_{\mathbf{B}} = \frac{-W_{1.}b_{1.}a_{1}^{2}}{L^{2}} - \frac{W_{2.}b_{2.}a_{2}^{2}}{L^{2}} = \frac{-Fx5.5x1.5^{2}}{7^{2}} - \frac{15x3x4^{2}}{7^{2}} = -0.25 \text{ F} - 14.93 $ (2)	01M
		To get value of 'F', use $M_A = M_B$ (Given)	01 M
		-0.92 F -11.02 = -0.25 F -14.93	
		0.67 F =3.91	
		F= 5.835 KN.	
		Explain the concept of fixity with effect in fixed beam	4 M
	d. Ans.	If simply supported beam is considered subjected to any pattern of loading, beam bends and slopes will be developed at the ends. If however, the ends of beam are firmly built in supports i.e. ends are fixed, slopes at the supports are zero. Fixity at ends induces end moments. Due to fixity, deflection of beam at center of beam is also reduced as compared to simply supported beam.	2 M
		Simply supported beam Fixed Beam	2 M
Q-4		Attempt any THREE of the following	12 M
	a.	Explain the concept of imaginary zero span in case of Clapeyron's theorem.	4 M
	Ans.	When the ends of the continuous beam are fixed, then an imaginary zero span is taken or considered to the left or right of the support as the case may be and the Clapeyron's theorem is applied to an imaginary span and its adjacent span. From the following Fig, the concept of zero span is well understood.	2 M
		Ao B Ao Lo <u>H</u> L B (Zero span).	1 M
		The end A is fixed, hence assume an imaginary span A0A, (called as zero span) to the left of A so as to apply Clapeyron's theorem for span A0A, and AB	1 M







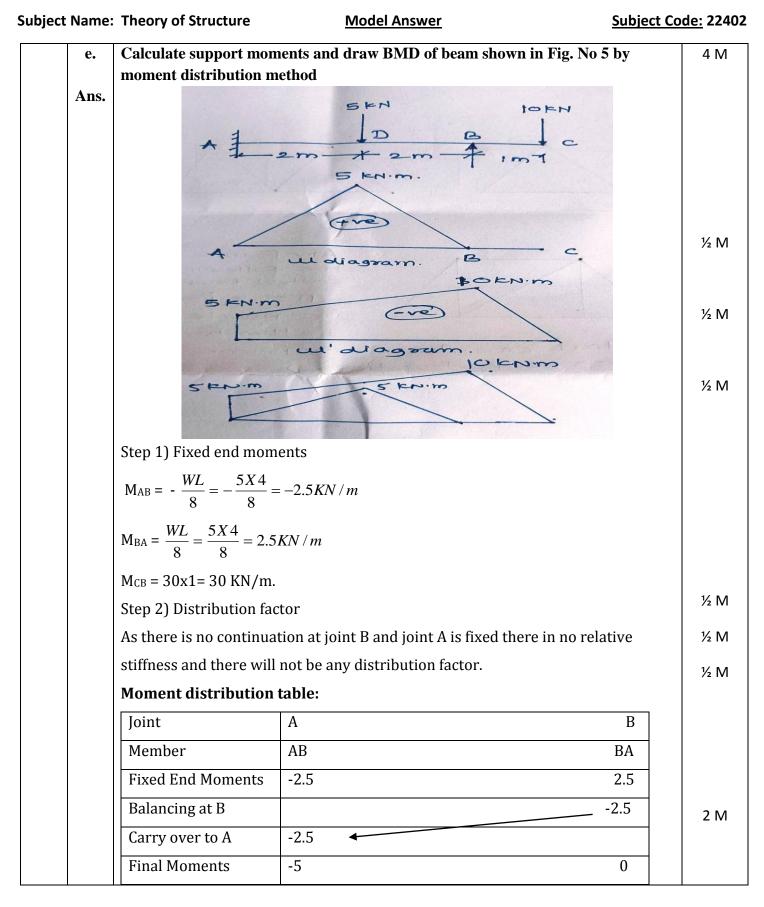




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$0 \times 4 + 2 \times M_{B} (4 + 5) + 0 \times 5 = -(\frac{6x133.33x2}{4} + \frac{6x90x2.33}{5}) = -651.64$	
M _B = - 36.20 KN.m	
Step 3) To find support reaction	1 M
For span AB,	
M @ B = $R_A X 4 - (25X4X2) + 36.20 = 0$	
R _A =40.95 KN.	
For span BC,	
M @ B =-36.20 + 30 x 3 - Rc x 5	
Rc = 10.76 KN	
Σ Fy = 0 = R _A + R _B + R _C - 25 X 4 - 30	
$40.95 + R_B + 10.76 = 130$	
R _B = 78.29 KN	
Step 4) Shear force calculation,	
S.F A _{Left} =0	1 M
S.F A _{Right} =40.95 KN	
S.F B _{Left} = 40.95-25 x4 =-59.05 KN.	
S.F B _{Right} = -59.05 +78.29 =19.24 KN.	
S.F D _{Left} = 19.24 KN	
S.F D _{Right} =19.24-30 =-10.76 KN	
S.F. C _{Left} =-10.76 KN.	
S.F @ C = -10.76 +10.76 =0 KN.	

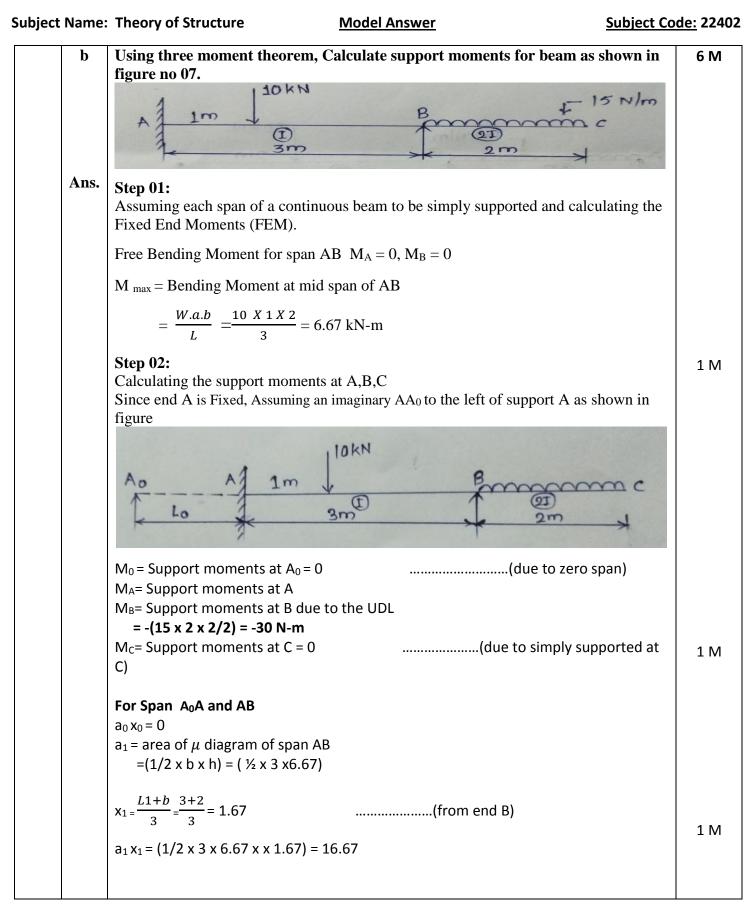






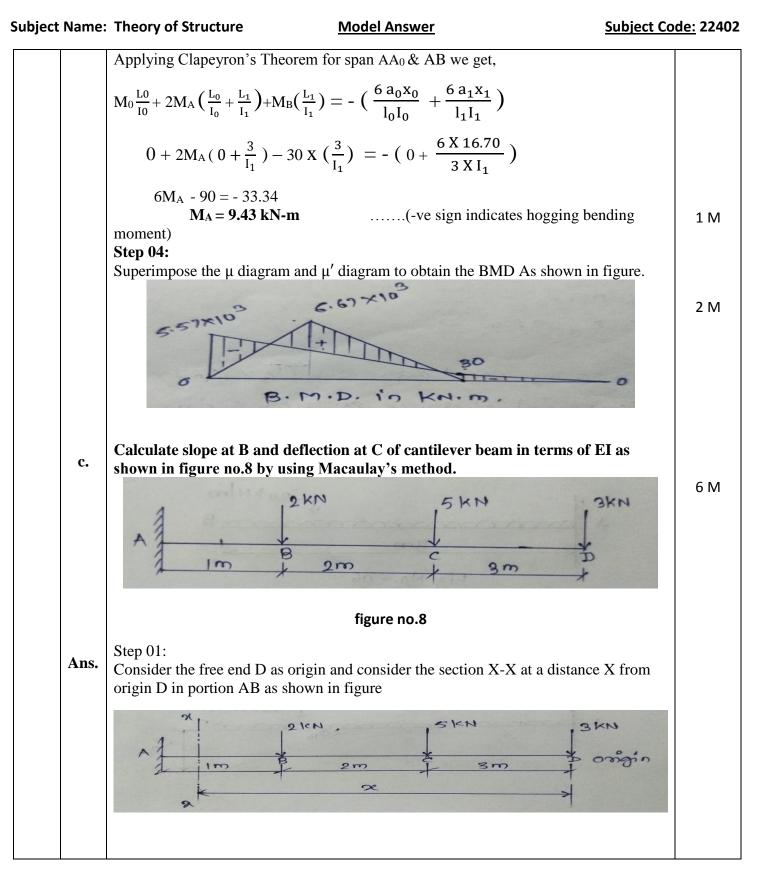
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		Fixed end moment at End A is = - 5 KN/m (- ve sign indicates hogging	
		moments)	
		Draw μ ' diagram for above value.	
		Steps 2) Calculate free end moments considering beam to be simply supported.	
		For span AB	
		$\mathbf{M}_{\max} = \frac{WL}{4} = \frac{5x4}{4} = \frac{5kN}{m}$	1 M
		Draw μ diagram for above value.	
Q-5		Attempt any Two of the following:	12 M
	a	Calculate slope and deflection at free end of cantilever beam as shown in figure	6 M
		no.06 having cross section 160 mm width and 220 mm depth. Using standard	
		formulae and take E=201 GPa	
		A Jamman B 3m	
	Ans.	Step 01: Moment of Inertia about X-X Axis $=\frac{bd^3}{12} = \frac{160 \times 220^3}{12}$	2 M
		$Ixx = 141.97 \times 10^6 \text{ mm}^4$	
		Step 02:	
		To find the slope at free end i.e. point B 20×3000^3	1 M
		$\boldsymbol{\theta B} = \boldsymbol{\theta max} = \frac{Wl^3}{6 EI} = \frac{20 X 3000^3}{6 X 201 X 10^3 X 141.97 X 10^6}$	1 M
		$\boldsymbol{\theta}\boldsymbol{B} = \boldsymbol{\theta}\boldsymbol{m}\boldsymbol{a}\boldsymbol{x} = 3.15 \text{ X } 10^{-3} \text{ radian}$	1 M
		Step 03: To find the deflection at free end i.e. point B. Let, $Y_B =$ Deflection at free end	
		$Y_{\rm B} = \frac{-Wl^4}{8 EI} = -\frac{20 X 3000^4}{8 X 201 X 10^3 X 141.97 X 10^6} = -7.09 \text{ mm (Downward)}$	1 M







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	The general bending moment equation at a distance X from D. (Considering, Clockwise moment +Ve & Antilock wise moment –ve)	
	(EI x $\frac{d^2 y}{dx^2}$) = Mx-x = (3.x) + 5 (x-3) + 2 (x-5)(i)	½ M
	Integrating above eq.(i) w. r. to x (EI x $\frac{dy}{dx}$) =3 $\frac{x^2}{2}$ + 5 $\frac{(x-3)^2}{2}$ + 2 $\frac{(x-5)^2}{2}$ + C ₁ (ii) (Slope Equation)	½ M
	Integrating above eq.(ii) w. r. to x (EI x y) = $\frac{3}{2} x \frac{x^3}{3} + \frac{5}{2} X \frac{(x-3)^3}{3} + \frac{2}{2} x \frac{(x-5)^3}{3} + C_1 x + C_2$	
	(EI x y) = $\frac{x^3}{2}$ + 2.50 X $\frac{(x-3)^3}{3}$ + $\frac{(x-5)^3}{3}$ + C ₁ .x + C ₂ (iii) (Deflection Equation)	1 M
	Step 02: Apply boundary condition for calculating the value of constant of integration C_1 and C_2 Condition 01:	
	At point A, where x= 6 m and $\frac{dy}{dx} = 0$ Put these value in eq.(i) i.e. slope equation $0 = (3x\frac{6^2}{2}) + (5x\frac{(6-3)^2}{2}) + (2x\frac{(6-5)^2}{2}) + C_1$	
	$0 = 54 + 22.50 + 1 + C_1$ C ₁ = - 77.50 kN	1 M
	At point A, where $x = 6$ m and $y = 0$	
	Put these value in eq.(ii) i.e. deflection equation	
	$0 = \left(\frac{6^3}{2}\right) + \left(2.50 \times \frac{(6-3)^3}{3}\right) + \left(\frac{(6-5)^3}{3}\right) - (77.50 \times 6) + C_2$	
	= 108+ 22.50 +0.33 -465 + C ₂	1 M
	C ₂ = 334.17	

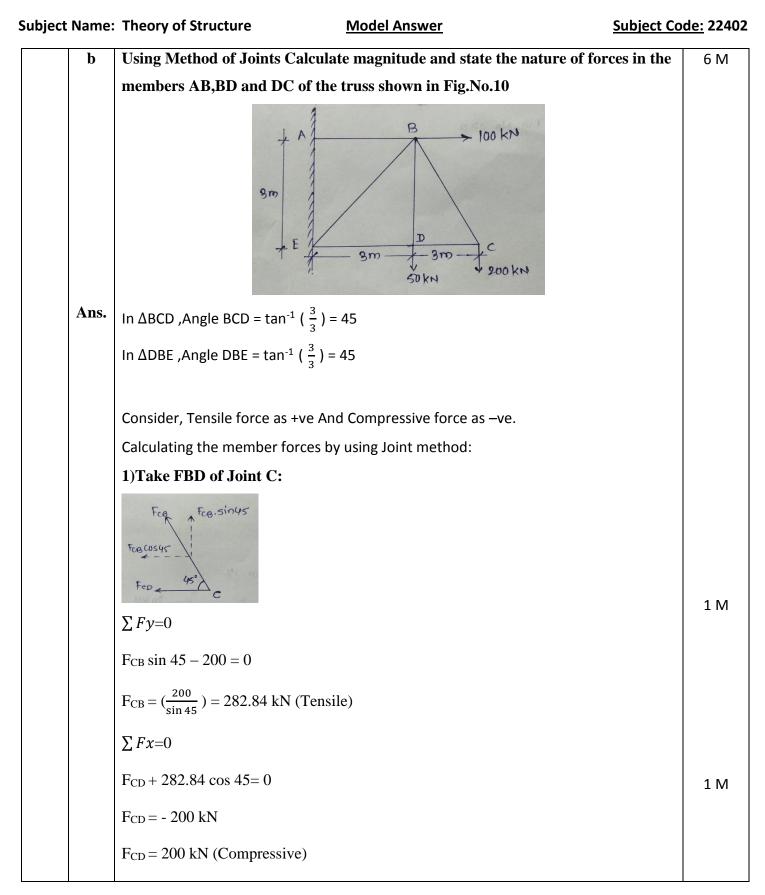


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		Step 03:	
		Calculating Slope at point B in terms of EI, where x= 5 m from origin D.	
		$(EIx\frac{dy}{dx})_{B} = 3\frac{5^2}{2} + 5\frac{(5-3)^2}{2} + 2\frac{(5-5)^2}{2} - 77.50$	
		$(EI \times \frac{dy}{dx})_B = 37.50 + 10 + 0.77.50$	1 M
		$\left(\frac{dy}{dx}\right)_{B} = \Theta B = -\frac{30}{EI}$	1
		Step 04:	
		Calculating deflection at point C in terms of EI, where x= 3 m from origin D.	
		$(EI \times y_c) = \frac{3^3}{2} + 2.50 \times \frac{(3-3)^3}{3} + \frac{(3-5)^3}{3} - (77.50 \times 3) + 334.17$	
		(El x y _c) = 13.50 + 0 + 0 - 232.50 + 334.17	
		$\mathbf{Y}\mathbf{c} = \frac{115.17}{EI}$	1 M
Q-6		Attempt any Two of the following:	12 M
	a.	Using moment distribution method, calculate the support moments of beam as shown in figure no.09	6 M
		A TOKNIM A TOKNIM A TOKNIM B DOKN D D D D D D D D D D D D D D D D D D D	
	Ans.	L1= 4 m , L2=6 m, W=10 kN/m W=20kN For span AB= 2I	
		For span BC = I	
		Step 01: Assuming each span of given beam to be fixed and calculating Fixed End Moments	
		(FEM)	
		For Span AB:	
		$M_{AB} = -\frac{WL_1^2}{12} = -\frac{10x4^2}{12} = -13.33 \text{ kN-m}$	
		$M_{BA} = + \frac{WL_1^2}{12} = -\frac{10x4^2}{12} = +13.33 \text{ kN-m}$	½ M
		12 12 12 12 12 12 12 12 12 12 12 12 12 1	½ M



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For Spa	an BD:					
M _{BD} = ·	$-\frac{Wab^2}{L^2} = -\frac{2}{2}$	$\frac{0x \ 4x 2^2}{6^2}$ = -8.89 kN-r	n			½ M
M _{DB} = -	$+ \frac{Wba^2}{L^2} = + \frac{2}{L^2}$	$\frac{20x\ 2x4^2}{6^2}$ = +17.78 kM	N-m			½ M
Step 0	2:					
To find	l stiffness fac	tor (K) for Joint B				
$K_{BA} = \frac{3}{2}$	$\frac{EI_1}{L_1} = \frac{3E.2I}{4} =$	1.50 EI				½ M
	$\frac{EI_2}{L_2} = \frac{3EI}{6} = 0.$					½ M
$\sum K = ($	1.50 El +0.50	EI) = 2.00 EI				
Step 0	3:					
Calcula	ating Distribu	tion Factor:				1/ 1/
(DF) _{BA} =	$=\frac{K_{BA}}{\sum K}=\frac{1.50\ I}{2\ EI}$	21 = 0.75				½ M
(DF) _{BC} =	$=\frac{K_{BC}}{\sum K}=\frac{0.50\ E}{2\ EI}$	 = 0.25				½ M
Check:	(0.75 + 0.25)	= 1.00				
Step 0	4: Moment D	istribution Table				
Joint		А	В		D	02 M
Mem	ber	АВ	BA	BD	DB	
Distri Facto	bution r		0.75	0.25		
FEM		-13.33	+13.33	-8.89	+17.78	
	se Support C.O. from A	+13.33			-17.78	
В	& From D to		+6.665	-8.890		
	Moments Distribution	0	+19.995	-17.780	0	
Balan	ce at B		-1.661	-0.554		
Final	Moments	0	+18.334	-18.334	0	

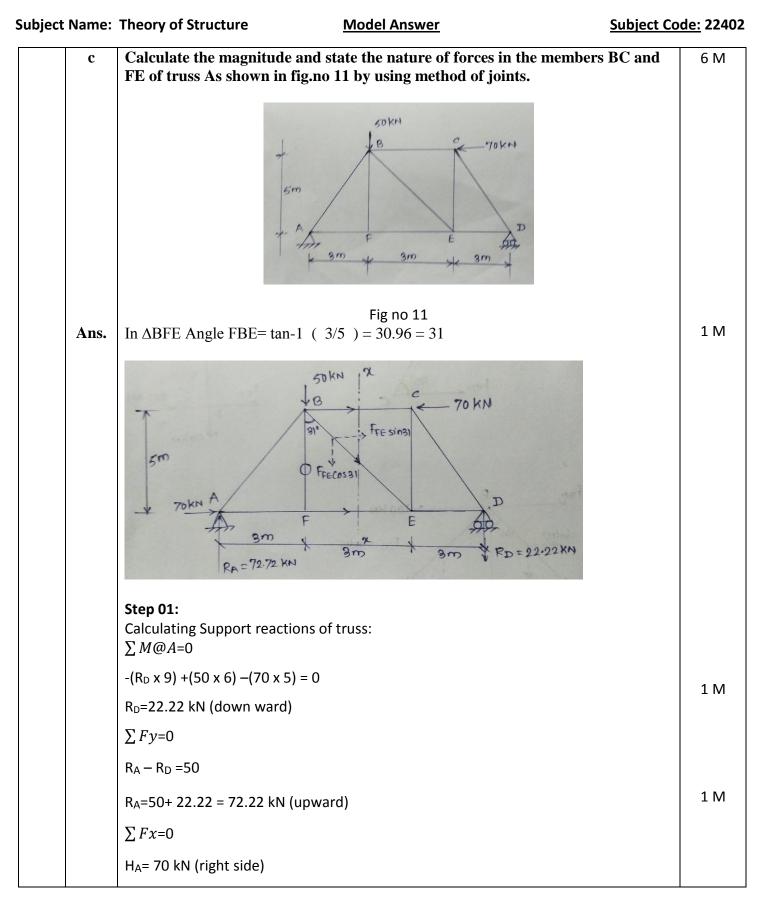






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	2) Tak	e FBD of joi	nt D:				
	Fpe <	ГDВ 200 Д 750 KN	KN				
	$\sum Fy =$	0					
	F _{DB} – 5						
	$F_{DB} = 5$	0 kN (Tensil	e)			1 M	
	$\sum Fx =$	0					
	$F_{DE} + 200 = 0$						
	$F_{DE} = -200 \text{ kN}$						
		00 kN (Com				1 M	
	3) Take FBD of Joint B:						
	$F_{BE} \xrightarrow{B} 100 \text{ kN}$ $F_{BE} \xrightarrow{\text{sin4s}} 4s^{\circ} \xrightarrow{\text{sin4s}} F_{CB} \xrightarrow{\text{sin4s}} 50 \text{ kN}$ $F_{BE} \xrightarrow{\text{sin4s}} F_{CB} \xrightarrow{\text{cos4s}} 50 \text{ kN}$						
	$\sum Fy=0$						
	$-50 - 282.84 \cos 45 - F_{BE} \cos 45 = 0$						
	$-250 - F_{BE} \cos 45 = 0$						
	$-F_{BE} = \frac{250}{\cos 45} = 353.55 \text{ kN}$						
	cos 45 F _{BE} = 353.55 kN (Compressive)						
	$\sum Fy=0$						
	F_{BA} + 353.55 sin 45 -100 – 282.84 sin 45 = 0						
	F _{BA} = 50 kN (Tensile)						
	Sr.	Member	Force	Nature			
	1	AB	50 kN	Tensile			
	2	BD	50 kN	Tensile			
	3	DC	200 kN	Compressive			
	-	-					







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	Considering the sectin X-X and applying conditions of equilibrium to the left part						
	truss.						
	∑ <i>M@</i>	<i>B</i> =0					
	-(F _{FE} x	5) + (72.22 >	(3) –(70 x 5)	= 0			
	F _{FE} = 2	6.68 kN (Tei	nsile)			1 M	
	$\sum Fy =$	0					
	$-50 + 72.22 - (F_{BE} \cos 31) = 0$						
	F _{BE} = 25.92 (Tensile)						
	$\sum Fx =$	0					
	70 + 26.68 + (25.92 sin31)+ $F_{BC} = 0$ $F_{BC} = -110.02 \text{ kN}$ $F_{BC} = 110.02 \text{ kN}(Compressive})$						
	Sr	Member	Force	Nature			
	1	BC	110.02 kN	Compressive			
	2	FE	26.68 kN	Tensile			